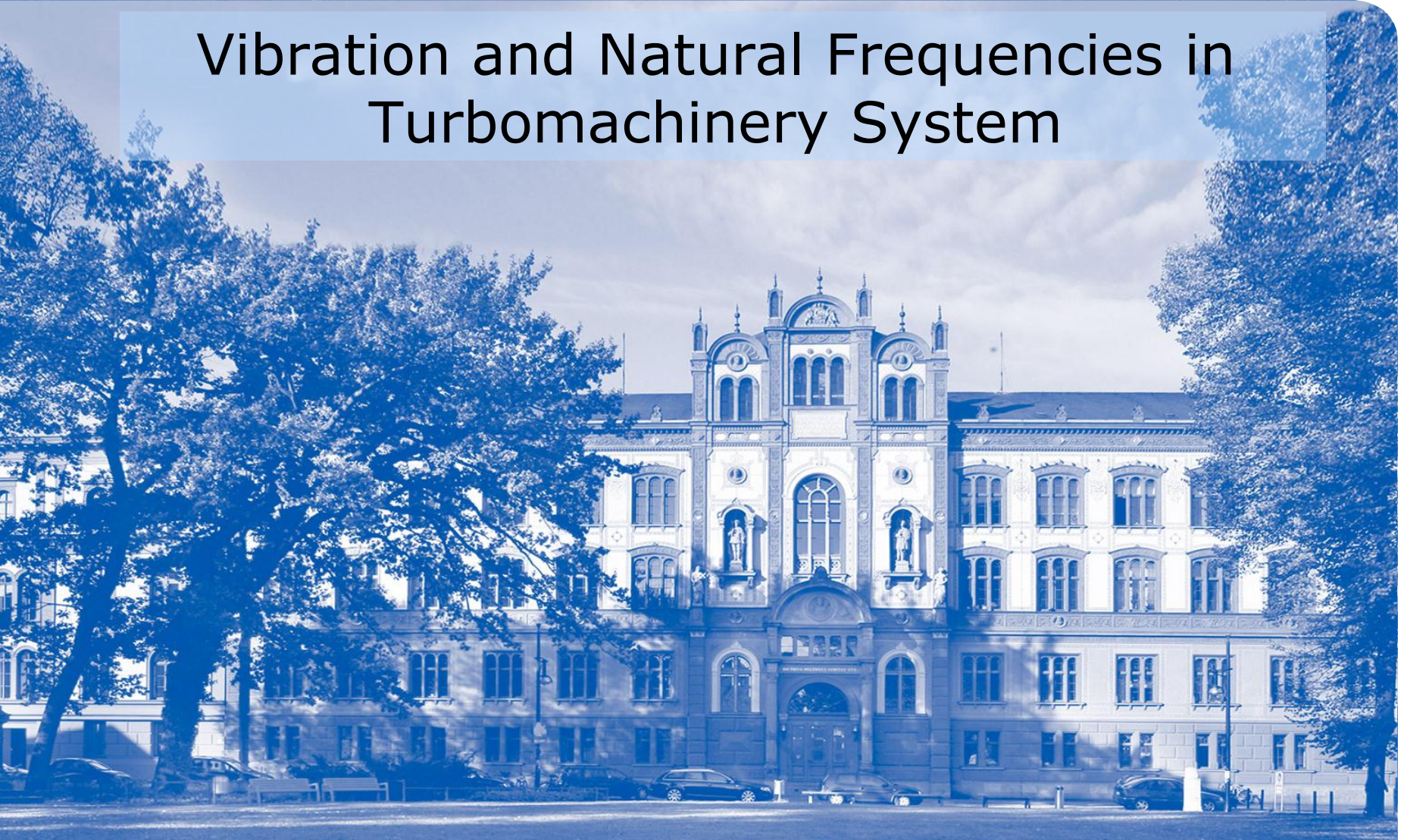


Vibration and Natural Frequencies in Turbomachinery System



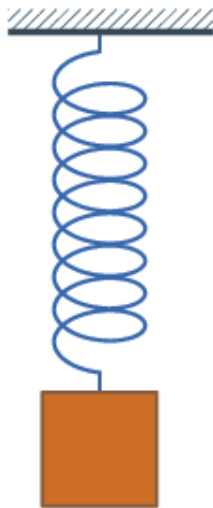


Outline

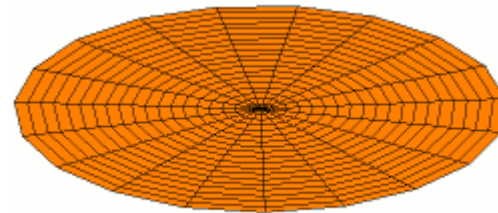
- Vibration
- Natural frequency
- Resonance
- Tacoma bridge
- Rotordynamics
- Numerical Techniques
- Vibration measurement

Vibration:

- Vibration is the result of energy being transferred back and forth between kinetic and potential energies. Vibration refers to mechanical oscillations about an *equilibrium point*.



Vibration of spring damper system



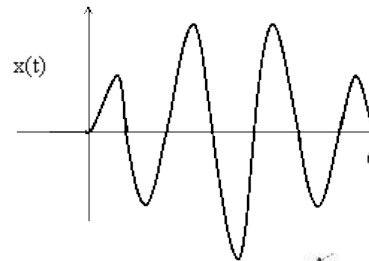
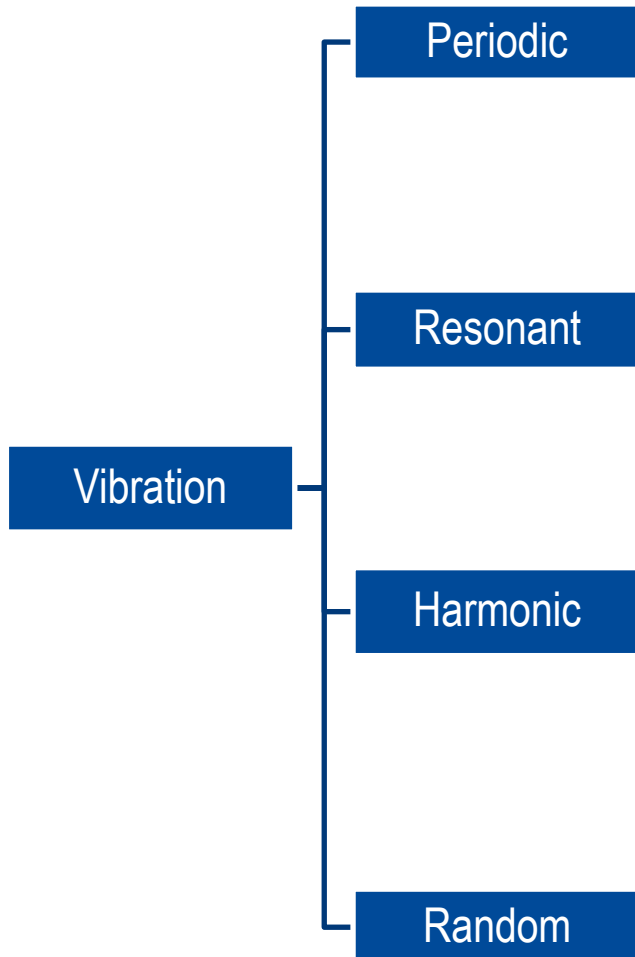
Vibration of circular disk



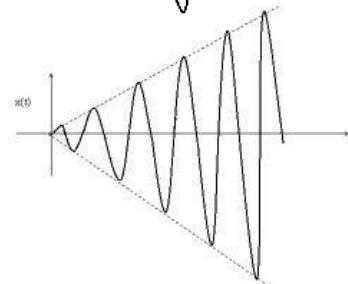
Why do we care about vibration?

- Vibration is wasted energy
- A major cause of premature component failure
- Cause of noise contribute to discomfort

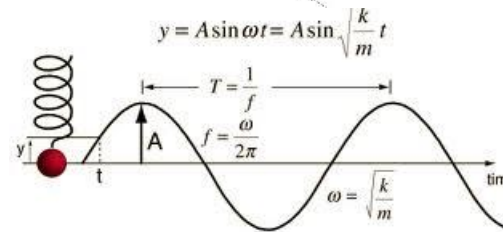
Vibration:



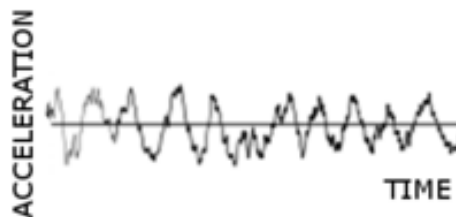
Example:
Rotation of disk having
mass imbalance



Example:
Tacoma bridge

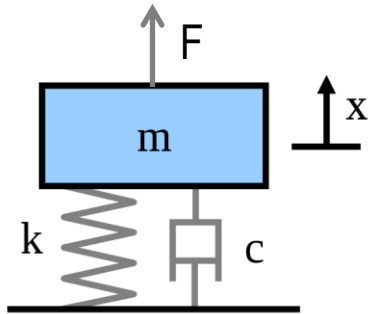


Example:
Pendulum or spring mass
system



Example:
the movement of a tire on
a gravel road.

General equation of Vibration:



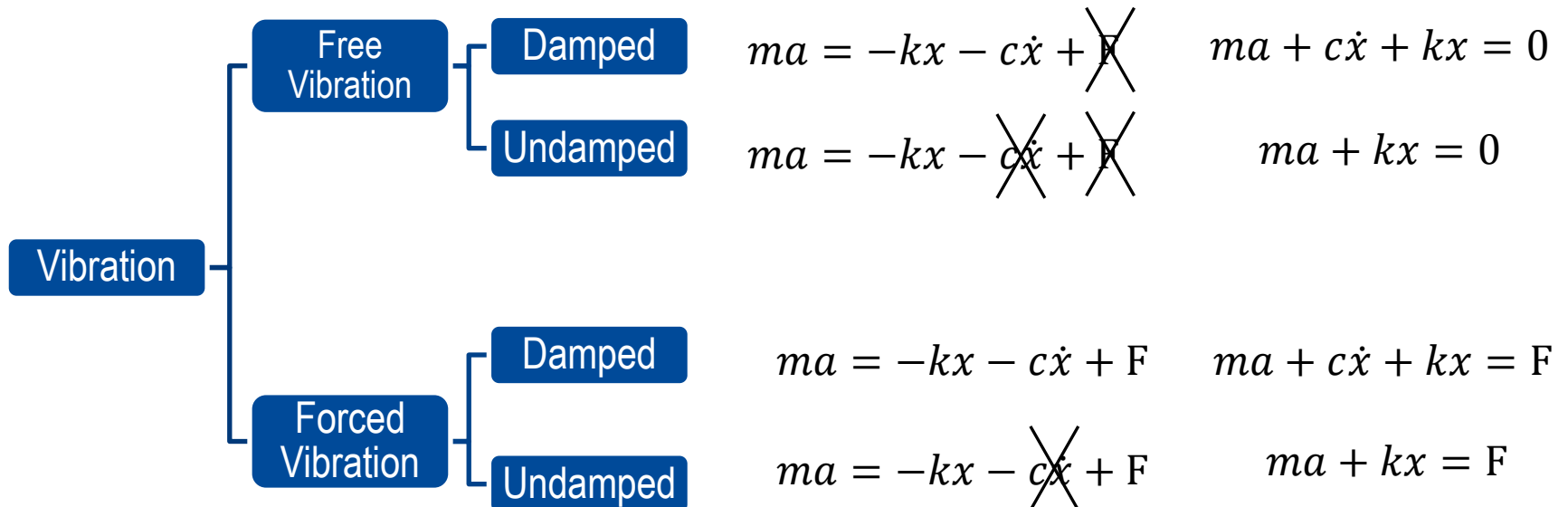
$$ma = -kx - c\dot{x} + F$$

k : Spring stiffness

c : Damping coefficient

F : Force

m : Mass



Damped Vibration:

$$ma + c\dot{x} + kx = 0$$

$$m\ddot{x} + c\dot{x} + kx = 0$$

$$\frac{d^2x}{dt^2} + 2\zeta\omega_0 \frac{dx}{dt} + \omega_0^2 x = 0$$

$$\zeta = \frac{c}{2\sqrt{mk}}$$

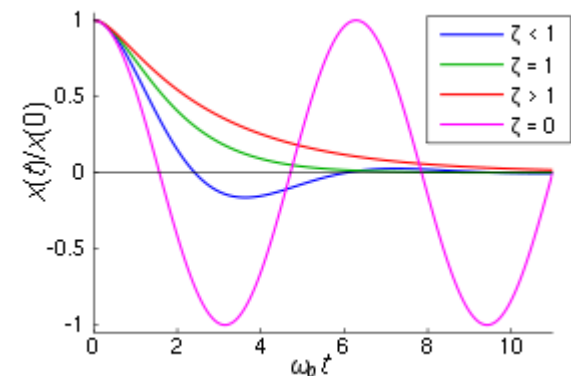
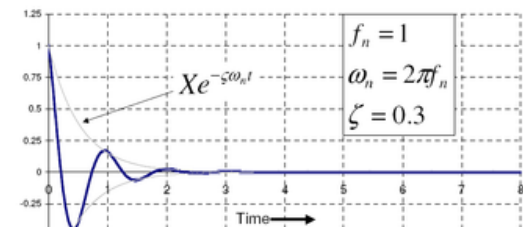
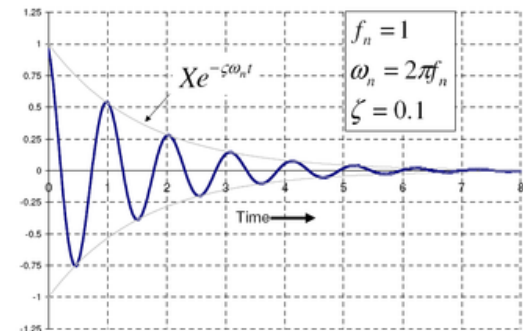
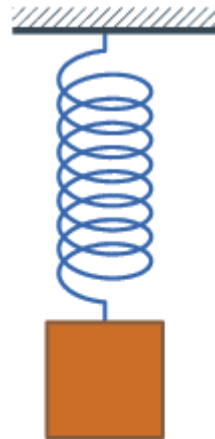
$$\omega_0 = \sqrt{\frac{k}{m}}$$

$\zeta = 1$: Critical damping

$\zeta > 1$: Over damped

$0 \leq \zeta < 1$: Under damped

ω_0 : Natural frequency



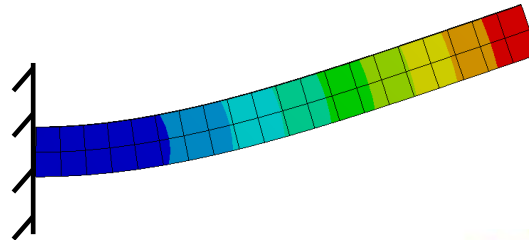
Natural modes and frequency:

- The natural frequency is the frequency at which a system oscillates when it is disturbed.
- Natural frequency depends of many factors, such as stiffness, mass, length and damping of material etc.
- The natural frequency is important for many reasons:
 - All things in the universe have a natural frequency, and many things have more than one.
 - If you know an object's natural frequency, you know how it will vibrate.
 - If you know how an object vibrates, you know what kinds of waves it will create.
 - If you want to make specific kinds of waves, you need to create objects with natural frequencies that match the waves you want.
 - Predict failure of mechanical structure because of vibration and resonance

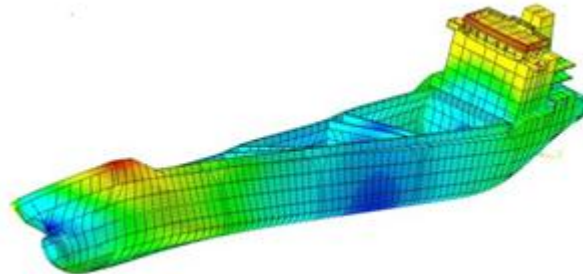


Type of modes:

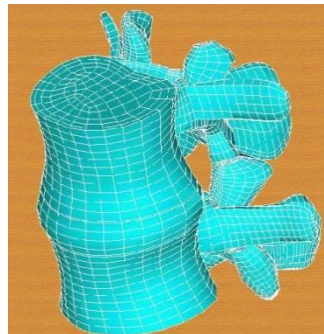
- Lateral bending modes



- Torsional modes



- Axial modes



Calculation of frequency:

$$m_1 \ddot{x}_1 + k_1(x_1 - x_2) = 0$$

$$m_2 \ddot{x}_2 + k_1(x_2 - x_1) + k_2 x_2 = 0$$

$$\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{Bmatrix} \ddot{x}_1 \\ \ddot{x}_2 \end{Bmatrix} + \begin{bmatrix} k_1 & -k_1 \\ -k_1 & k_1 + k_2 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

Let

$$x_1 = a_1 \sin wt \quad \ddot{x}_1 = -a_1 w^2 \sin wt$$

$$x_2 = a_2 \sin wt \quad \ddot{x}_2 = -a_2 w^2 \sin wt$$

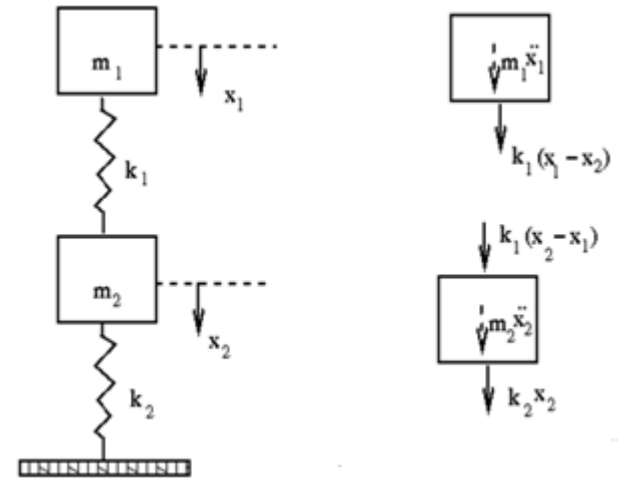
$$\begin{bmatrix} k_1 - m_1 w^2 & -k_1 \\ -k_1 & k_1 + k_2 - m_2 w^2 \end{bmatrix} \begin{Bmatrix} a_1 \\ a_2 \end{Bmatrix} = 0$$

$$\begin{vmatrix} k_1 - m_1 w^2 & -k_1 \\ -k_1 & k_1 + k_2 - m_2 w^2 \end{vmatrix} = 0$$

Assume:

$$k_1 = 10 ; m_1 = 5$$

$$k_2 = 20 ; m_2 = 10$$



Solving determinant we will get frequency of vibration.

By getting ratio of a_1 a_2 we can predict mode shape of vibration

Find frequency and modes.



Modal analysis using FEM:

$$[M] [\ddot{U}] + [C] [\dot{U}] + [K] [U] = [F]$$

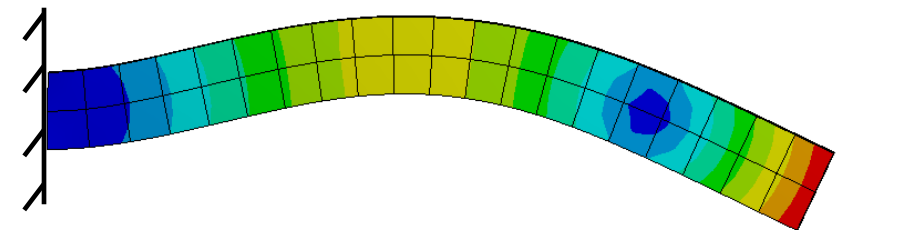
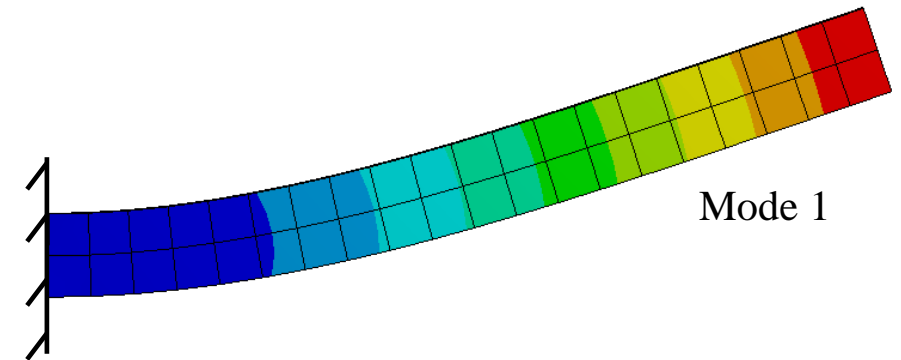
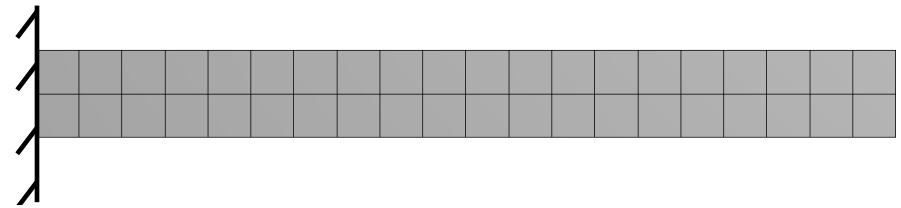
$$[M] [\ddot{U}] + [K] [U] = [0]$$

$$[M] = \begin{bmatrix} m_{11} & \cdots & m_{1n} \\ \vdots & \ddots & \vdots \\ m_{n1} & \cdots & m_{nn} \end{bmatrix}$$

$$[K] = \begin{bmatrix} k_{11} & \cdots & k_{1n} \\ \vdots & \ddots & \vdots \\ k_{n1} & \cdots & k_{nn} \end{bmatrix}$$



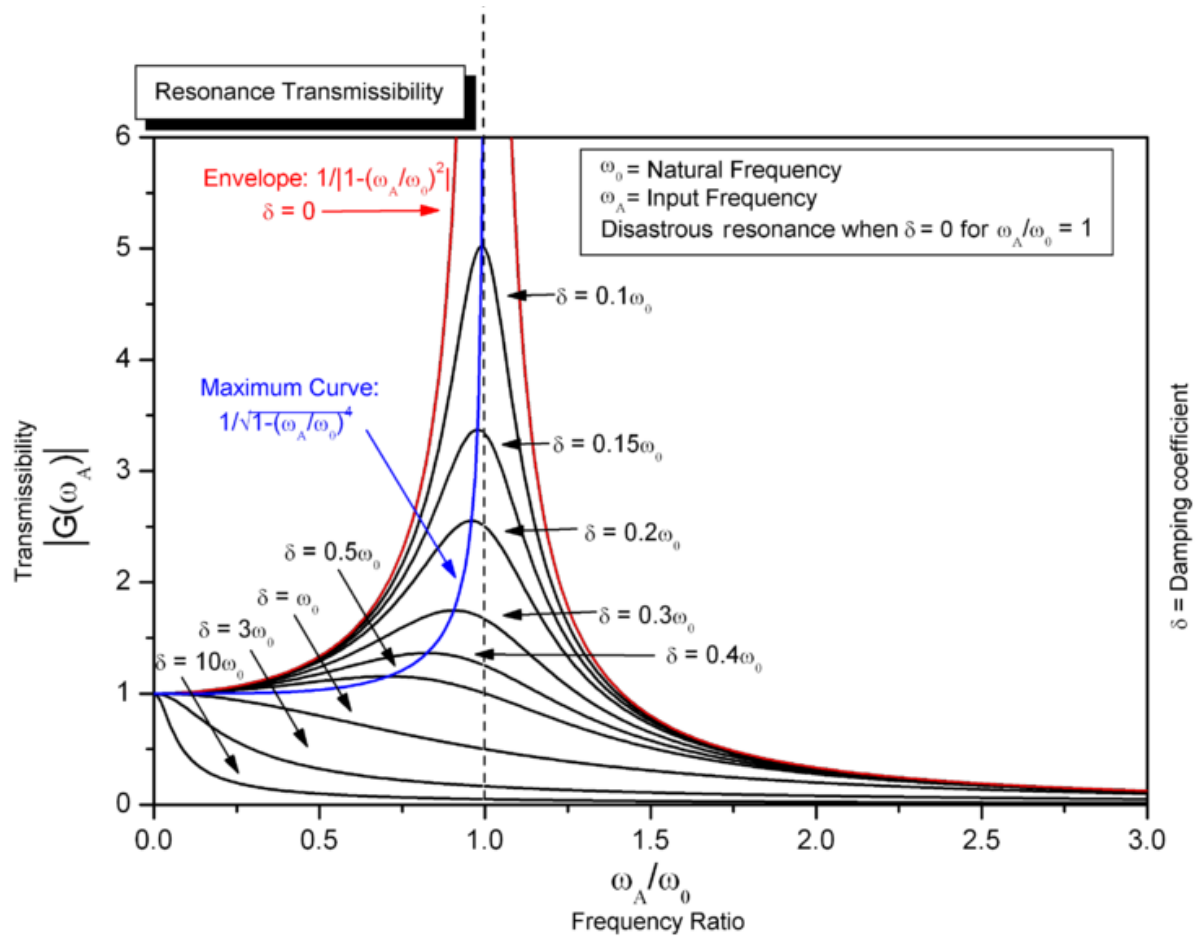
1. How many natural frequencies are possible of given system?
2. Which natural frequency is important?



Resonance:

- **Resonance** is the tendency of a system to oscillate with larger amplitude at some frequencies than at others.
- **Mechanical resonance** is the tendency of a mechanical system to absorb more energy when the frequency of its oscillations matches the system's natural frequency of vibration than it does at other frequencies.
- It may cause violent swaying motions and even catastrophic failure in improperly constructed structures including bridges, buildings, trains, and airplanes.
- Engineers when designing objects must ensure that the mechanical resonant frequencies of the component parts do not match driving vibrational frequencies of the motors or other oscillating parts a phenomenon known as resonance disaster.

Resonance:



Tacoma Bridge: Example

Start of construction: Nov,23,1938

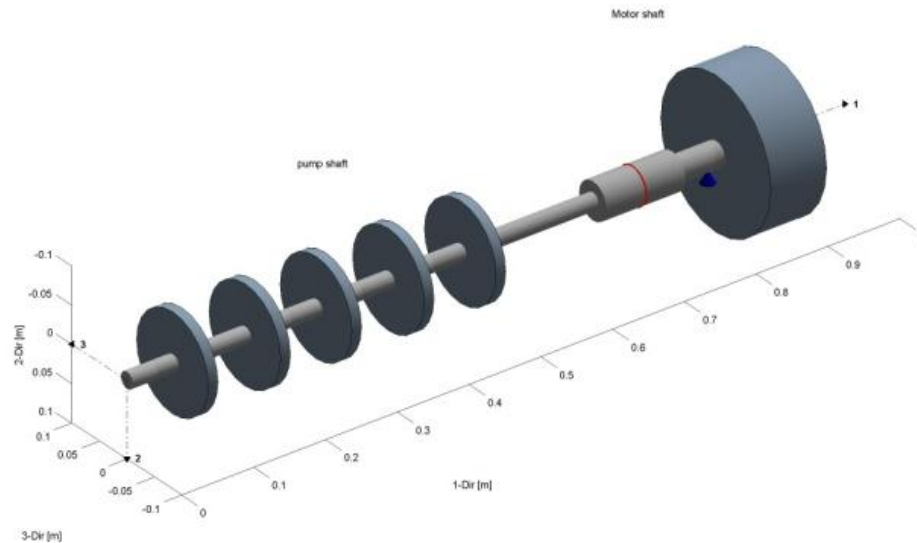
Opened for Traffic: July 1, 1940

Collapse of bridge: Nov,7,1940



Vibration in Turbomachinery system:

- Mechanically Induced
 - Bent Shaft
 - Unbalanced Rotor
 - Bad Bearings
 - Mis-Alignment
 - Looseness

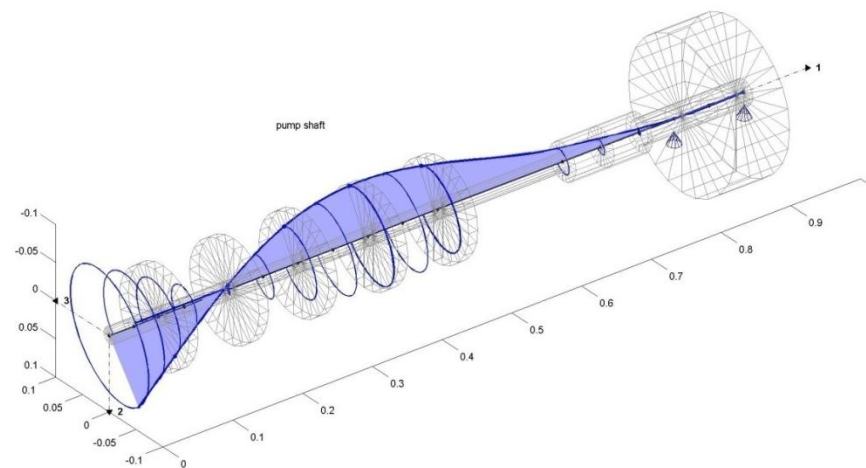
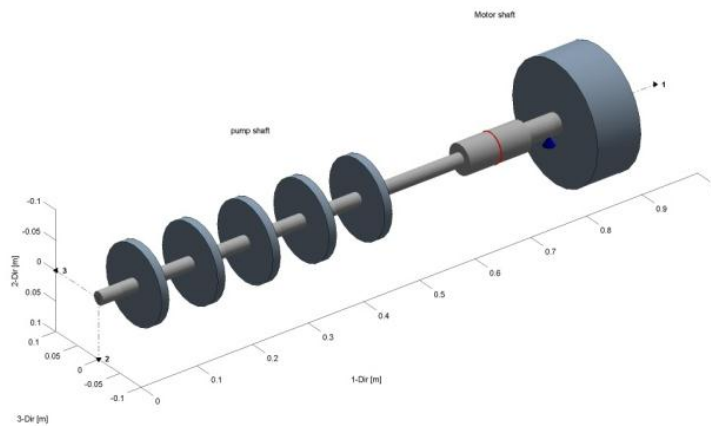


Vibration in Turbomachinery system:

- Operation Induced
 - Cavitation
 - Flow
 - Speed
 - Insufficient Immersion of suction pipe
- System Induced
 - Piping
 - Clogged impeller or Suction Line
 - Baseplate

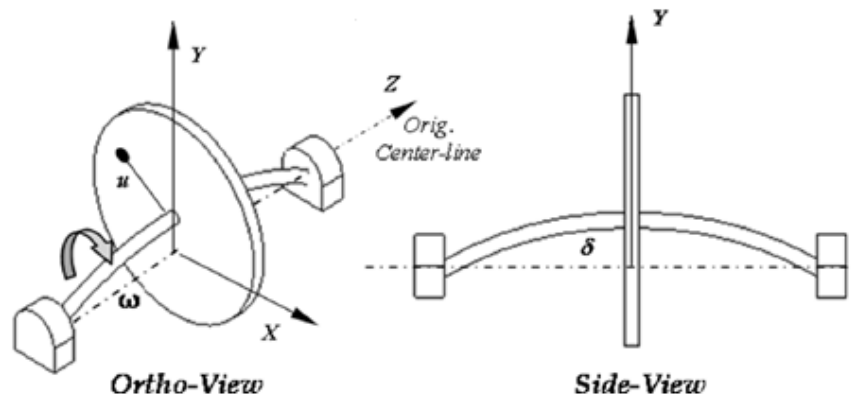
Rotordynamics:

- **Rotordynamics** is a specialized branch of applied mechanics concerned with the behavior and diagnosis of rotating structures.
- It is commonly used to analyze the behavior of structures ranging from jet engines and steam turbines to auto engines and computer disk storage.



Rotordynamics:

- At its most basic level rotordynamics is concerned with one or more mechanical structures (rotors) supported by bearings and influenced by internal phenomena that rotate around a single axis.
- The supporting structure is called a stator. As the speed of rotation increases the amplitude of vibration often passes through a maximum that is called a critical speed.
- This amplitude is commonly excited by unbalance of the rotating structure everyday examples include engine balance.
- If the amplitude of vibration at these critical speeds is excessive then catastrophic failure occurs.
- This is the chief concern of engineers who design large rotors.



General equation of motion:

$$M\ddot{q}(t) + (C + G)\dot{q}(t) + (K + N)q(t) = f(t)$$

where:

M : Symmetric Mass matrix

C : Symmetric damping matrix

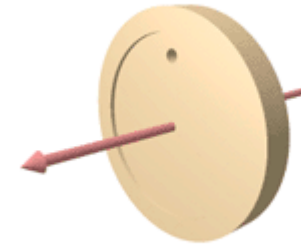
G : Skew-symmetric gyroscopic matrix

K : Symmetric bearing or seal stiffness matrix

N : Gyroscopic matrix of deflection.

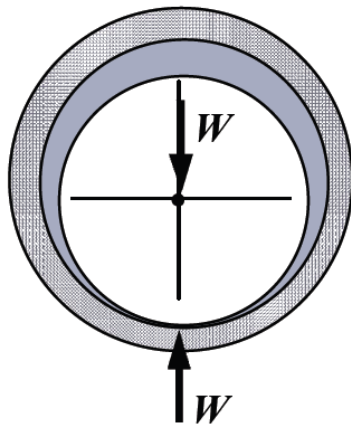
q : Generalized coordinates of the rotor in inertial coordinates

f : Forcing function.

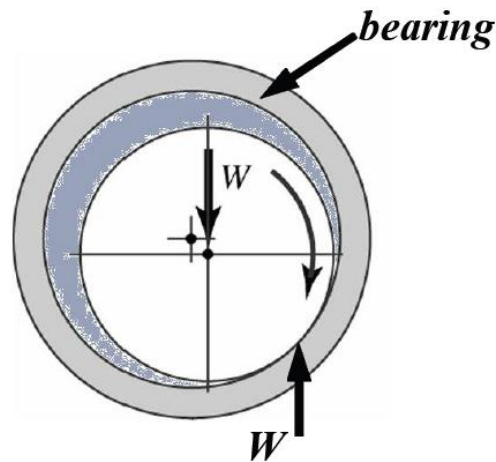


- The gyroscopic matrix G is proportional to spin speed Ω . The general solution to the above equation involves complex eigenvectors which are spin speed dependent.
- Rotordynamic system of equations are the off-diagonal terms of stiffness, damping, and mass. These terms are called cross-coupled stiffness, cross-coupled damping, and cross-coupled mass

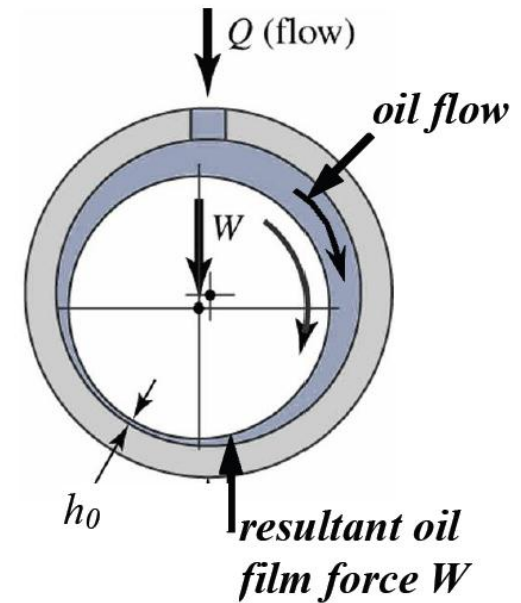
Hydrodynamic Bearings:



(a) At rest



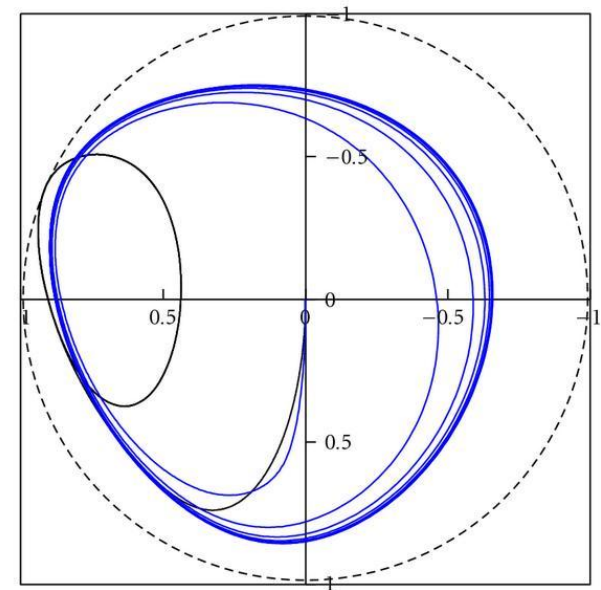
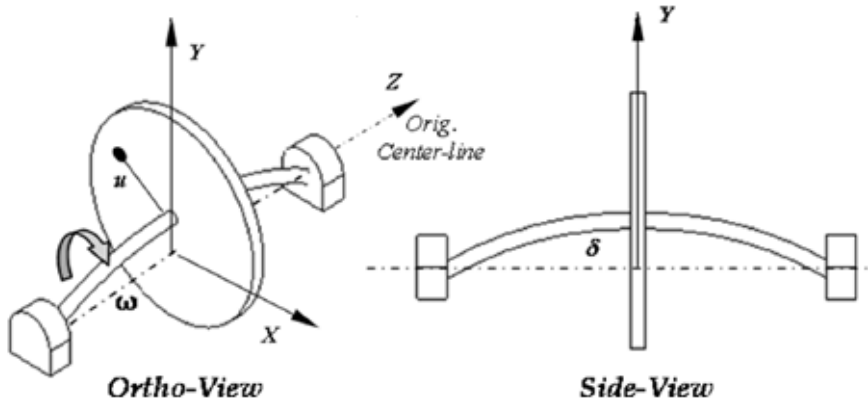
(b) Slow rotation
(Boundary lubrication)



(c) Fast rotation
(Hydrodynamic lubrication)

Whirling:

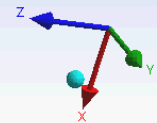
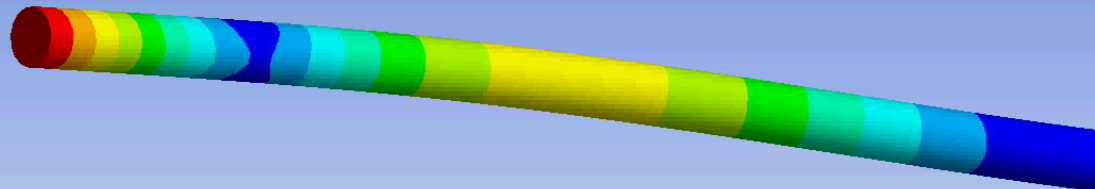
- Whirling is defined as a rotation rapidly about a center or an axis.



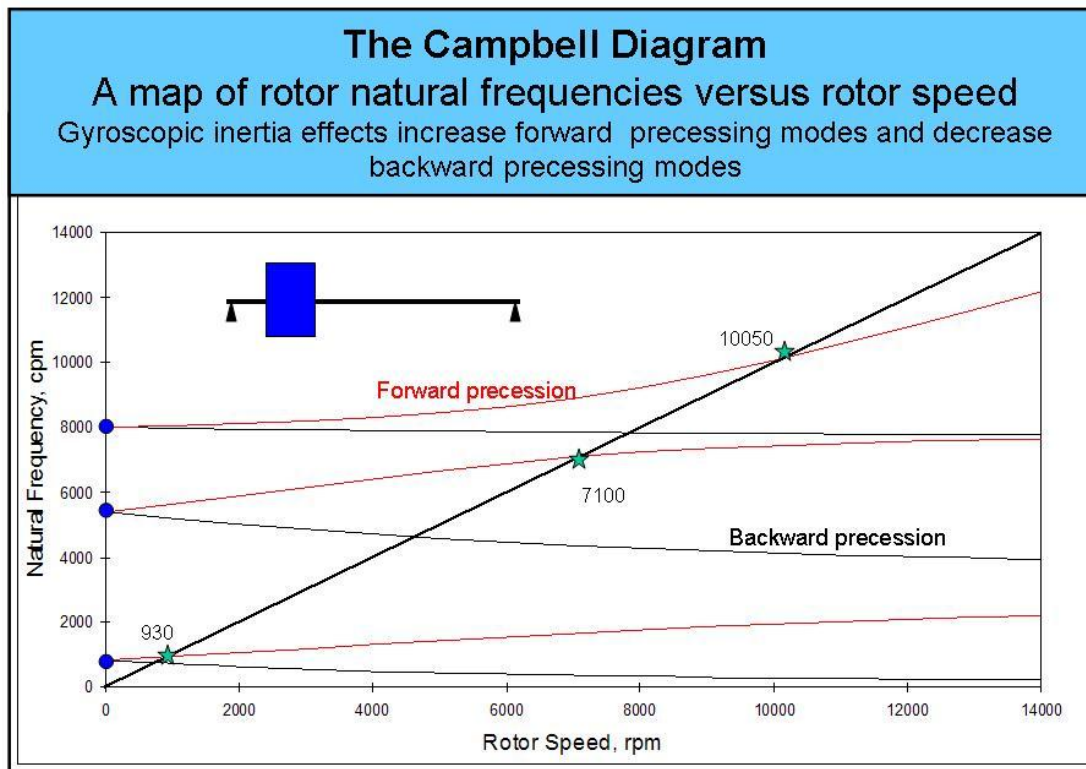
1. What is backward whirling?
2. What is forward whirling?

Whirling: Example

B: Modal
Total Deformation 2
Type: Total Deformation
Frequency: 1084.7 Hz
Phase Angle: 0.°
Unit: m
6/9/2011 2:38 PM



Campbell diagram:

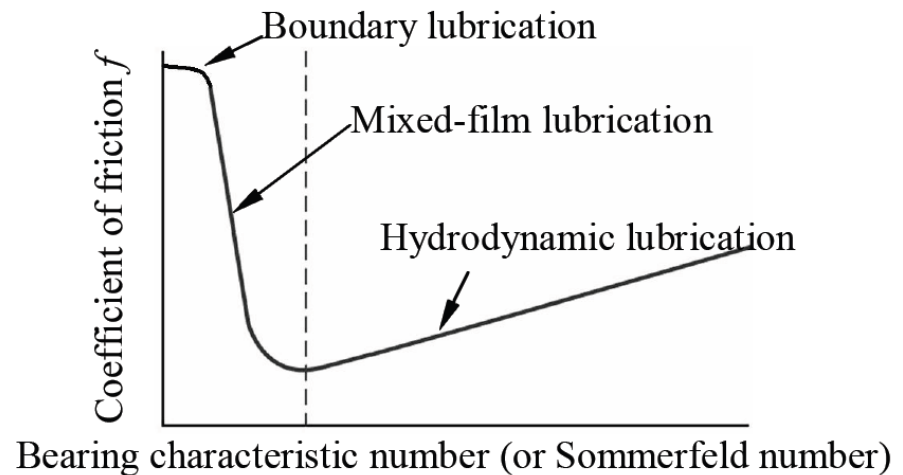
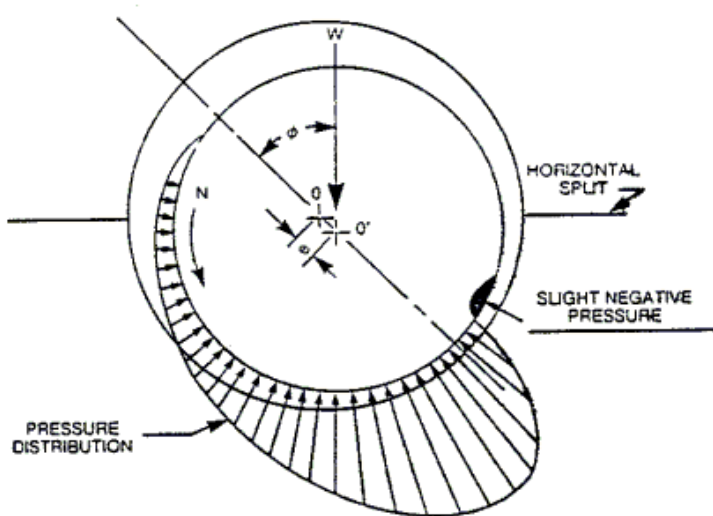


1. What is safety criteria?

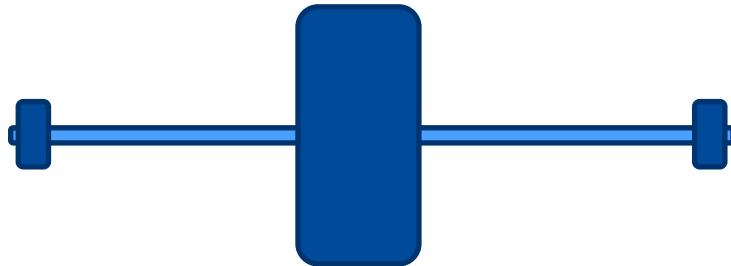
Sommerfeld number:

$$S = \left(\frac{r}{c}\right)^2 \frac{\mu \cdot n \cdot D \cdot L}{W}$$

W = total weight
 r = shaft diameter
 c = clearance
 n = rotational speed
 μ = viscosity
 D = shaft diameter
 L = bearing length



Example:



Shaft Diameter, D : 4 mm

Bearing length, L : 8 mm

Diametric Clearance, C : 1mm

Lubricant : Water

Viscosity : 8.9×10^{-4} Pa.s

Density : 1000 kg/m^3

Rotational Speed : 3000 RPM

Weight of rotor per bearing : 1.77 N

$$S = \frac{\mu N \pi D L}{W} \left(\frac{R}{C} \right)^2$$

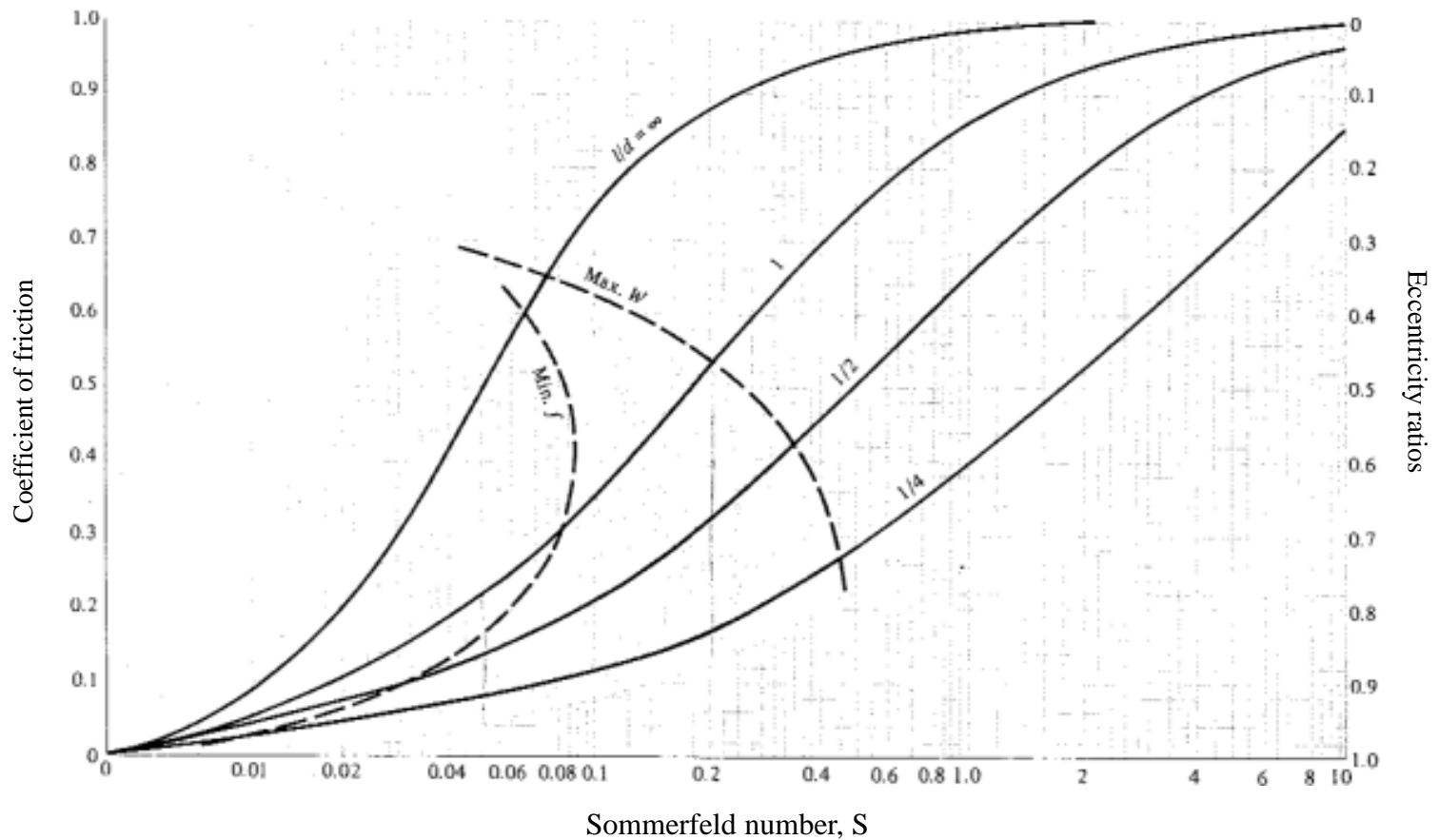
$$S = \frac{3.14}{1.77} \times 8.9 \times 10^{-4} \times (3000/60) \times 4 \times 8 \times 10^{-6} \left(\frac{2}{0.5} \right)^2$$

$$S = 4.02 \times 10^{-5} \text{ (very low)}$$

$\epsilon \geq 0.99$ (very high) (Predicted based on Raimondi and boyd chart of bearing)

$$h_{min} = C(1 - \epsilon) = 0.5(1 - 0.99) = 5 \times 10^{-3} \text{ mm (very low)}$$

Raimondi and boyd chart:

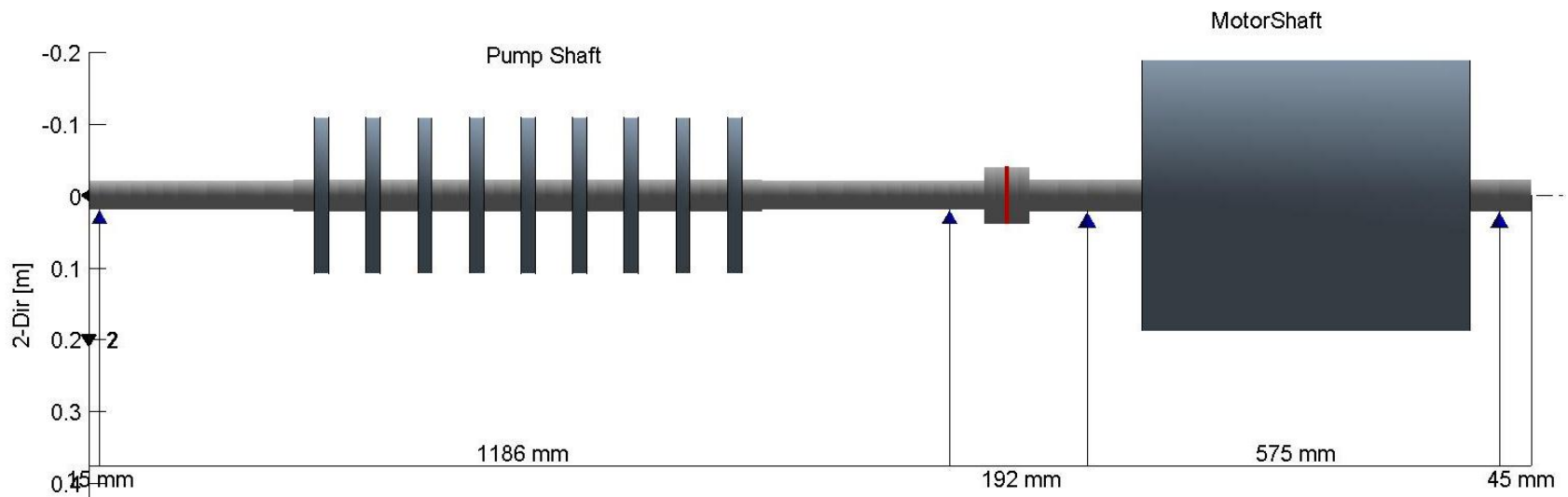




Software available for Rotordynamics:

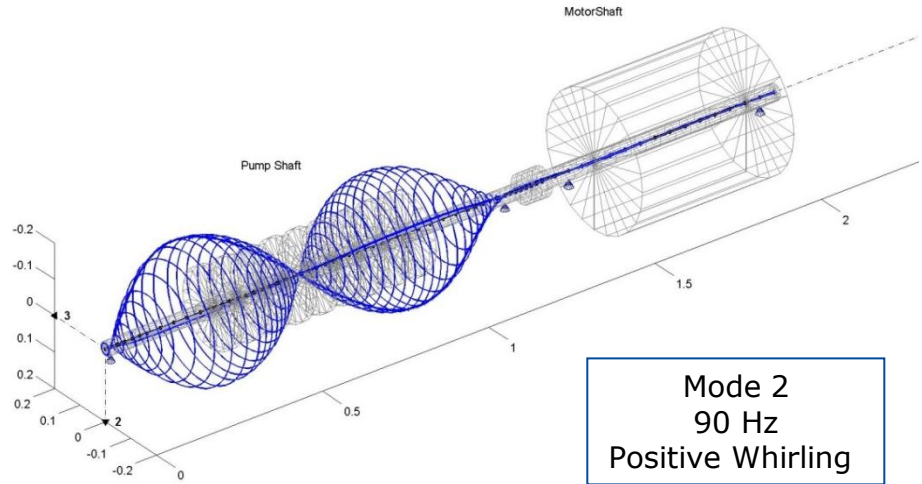
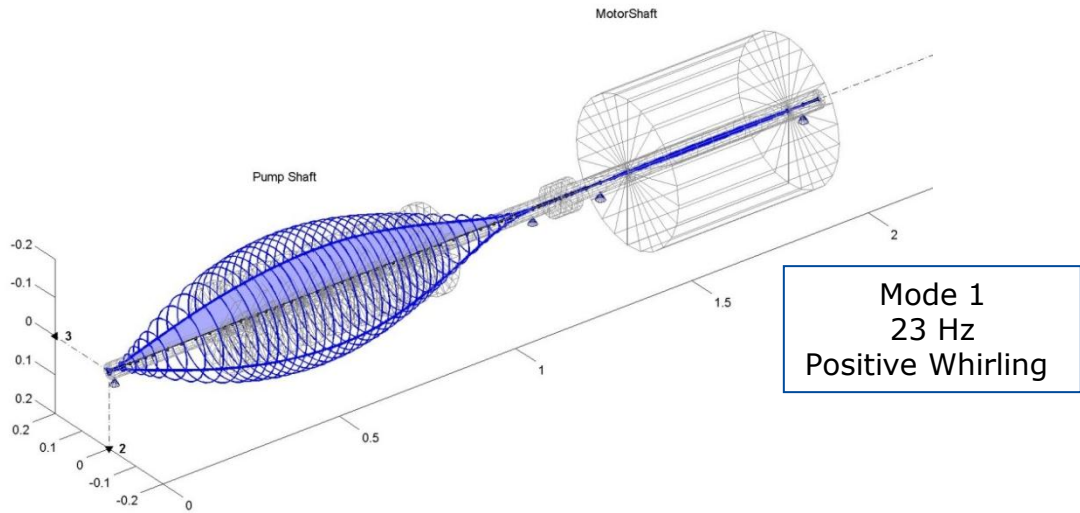
- MADYN 2000
- Bond Graph
- Dyrobes
- RIMAP
- XLRotor
- iSTRDYN
- ARMD
- XLTRC2
- ComboRotor
- Dynamics R4
- MESWIR
- RoDAP
- ROTORINSA

MADYN:

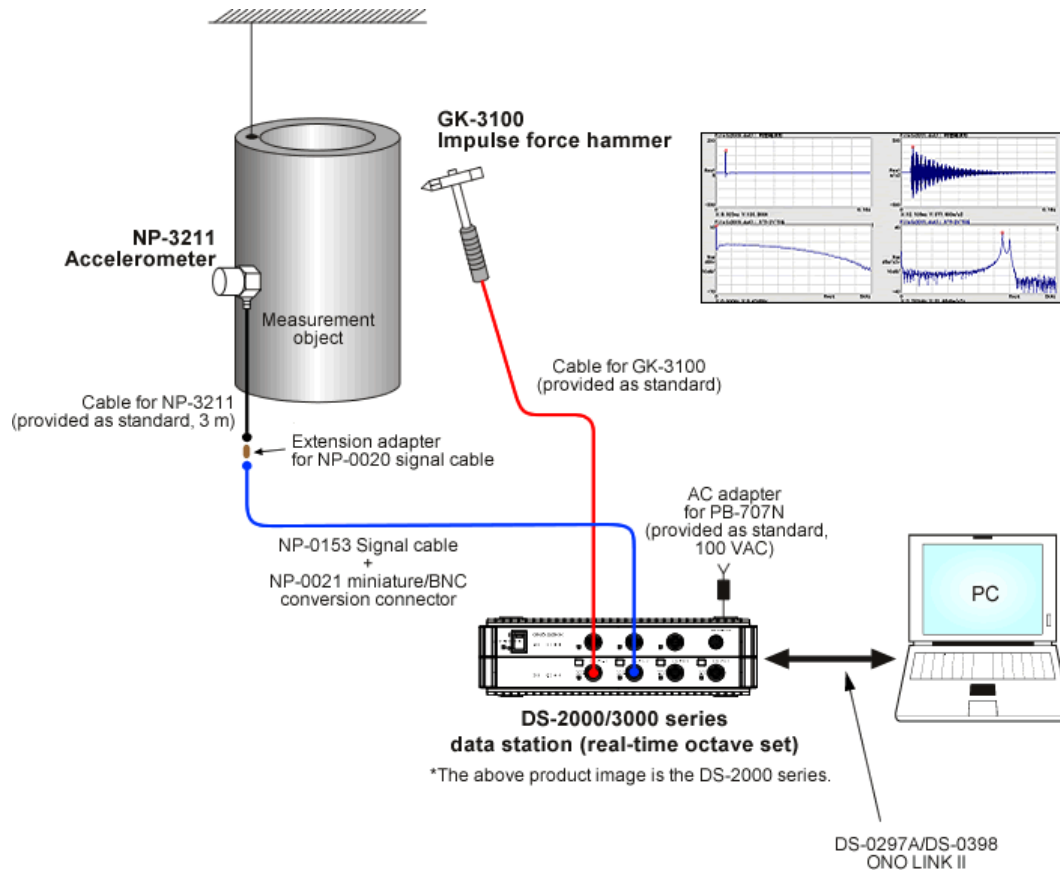


MADYN:

Operating Frequency
48.33 Hz

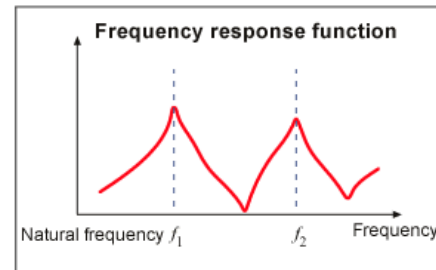
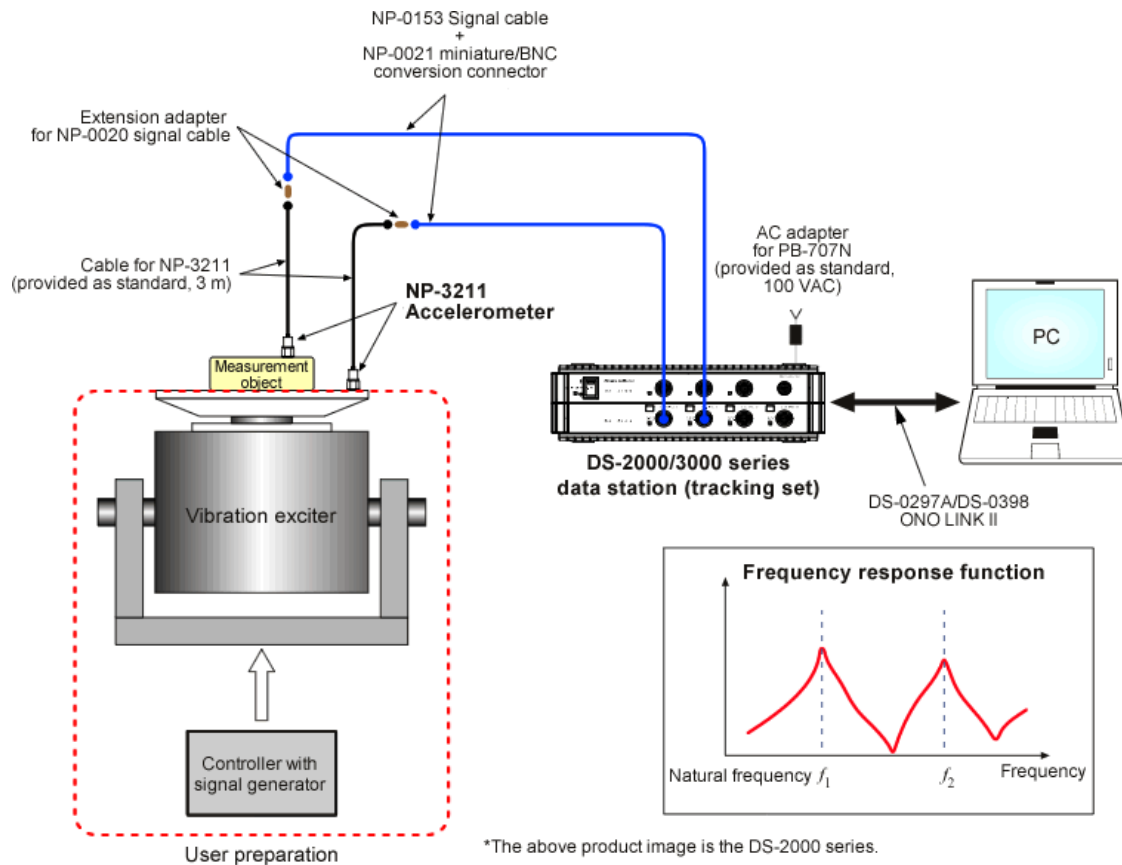


Vibration Testing:



Measurement of natural frequency and damping ratio by Hammer Testing

Vibration Testing:



*The above product image is the DS-2000 series.

Measurement of natural frequency using exciter controller



Thank You